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Heat transfer model of large shipping containers

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March 2013

Outline:

I. The heat transfer model

- 1. Heat transfer at the wall of the shipping container**
- 2. Heat transfer from the wall to the inside air**
- 3. Heat transfer at the cargo on the pallets**

II. Case Study: Semi Trailer of an eighteen wheeler

$$\begin{aligned}
 M_w \cdot C_{p_w} \cdot \frac{\Delta T_w}{dt} = & \underbrace{h_{FC} \cdot A \cdot (T_{air.out} - T_w)}_{\text{Forced convective heat transfer}} + \underbrace{h_{NCout} \cdot A \cdot (T_w - T_{air.out}) + h_{NCin} \cdot A \cdot (T_{air.in} - T_w)}_{\text{Natural convective heat transfer}} + \\
 & \underbrace{\frac{k \cdot A \cdot (T - T_w)}{\Delta x}}_{\text{Conductive heat transfer}} + \underbrace{A \cdot \alpha_w \cdot G_{solar}}_{\text{Solar radiation heat transfer}} + \underbrace{A \cdot \varepsilon \cdot \sigma (T_{Sky}^4 - T_w^4)}_{\text{Atmospheric radiation}}
 \end{aligned}$$

Forced convective heat transfer

$$Re = \frac{V \cdot \rho \cdot L}{\mu}$$

Laminar Flow: $Re < 10^5$

$$Nu = 0.664 Re^{0.5} Pr^{1/3}$$

Turbulent Flow: $Re > 10^5$

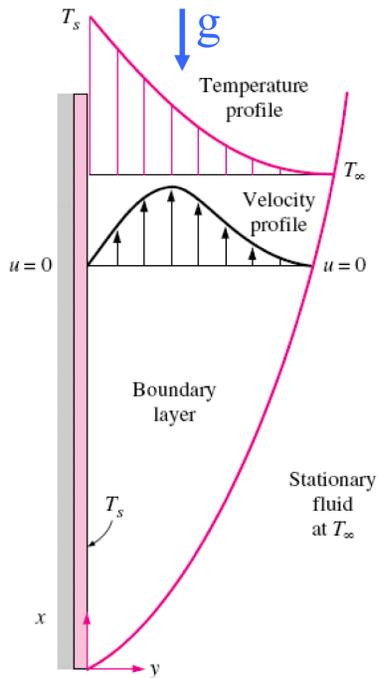
$$Nu = 0.037 Re^{0.8} Pr^{1/3}$$

$$h_{FC} = \frac{Nu \cdot k}{L}$$

Natural convective heat transfer

Rayleigh number

$$Ra_L = \frac{g \cdot \beta \cdot (T_s - T_{air}) L_c^3}{\nu^2} Pr$$



Vertical walls

$$Nu = \left\{ 0.825 + \frac{0.387 \cdot Ra_L^{1/6}}{\left[1 + (0.492/Pr)^{9/16} \right]^{1/4}} \right\}^2$$

Horizontal wall - upper surface of a hot wall

$$Nu = 0.54 \cdot Ra_L^{1/4} \quad \text{For } 10^4 < Ra_L < 10^7$$

$$Nu = 0.15 \cdot Ra_L^{1/3} \quad \text{For } 10^7 < Ra_L < 10^{11}$$

Horizontal wall - lower surface of a hot wall

$$Nu = 0.27 \cdot Ra_L^{1/4} \quad \text{For } 10^5 < Ra_L < 10^{11}$$

Conductive heat transfer

$$\frac{k \cdot A \cdot (T - T_w)}{\Delta x}$$

Δx - thickness of the insulation or layer of air near the wall

k - thermal conductivity coefficient

$$Nu = 0.27 \cdot Ra_L^{1/3}$$

Solar radiation heat transfer

$$A \cdot \alpha_s \cdot G_{solar}$$

G_{solar} - total solar irradiance [W/m^2]

α_s - solar absorptivity

The solar radiation incident on the Earth's surface has two components: direct solar radiation (G_D) and diffuse solar radiation (G_d).

$$G_{solar} = G_D + G_d$$

Direct solar radiation is computed using a solar model

$$f(t, Lat, Lon, Altitude)$$

Diffuse radiation represents around 10% of the global radiation on a clear sky day.

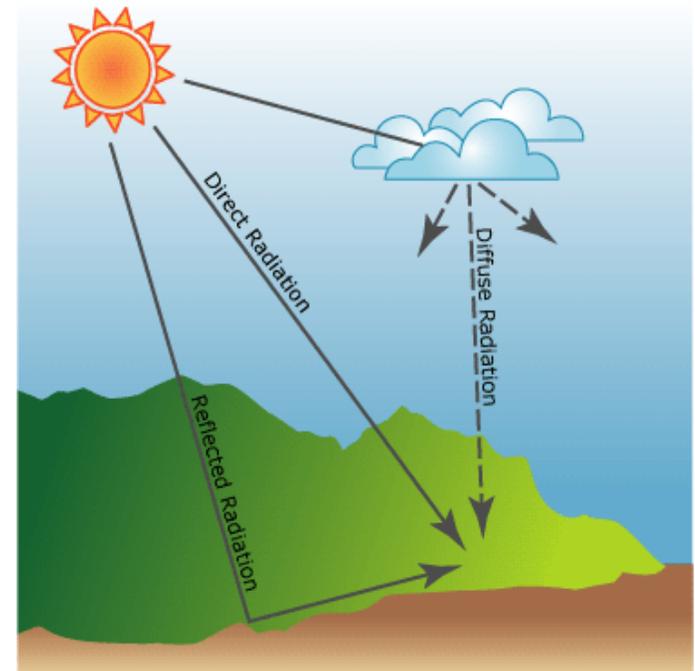
Atmospheric radiation

$$A \cdot \varepsilon \cdot \sigma (T_{sky}^4 - T_w^4)$$

ε - surface emissivity

σ - Stefan–Boltzmann constant

$5.670373 \cdot 10^{-8} [W/m^2 \cdot K^4]$



$$M_{air} \cdot C_{p_{air}} \cdot \frac{\Delta T_{air}}{dt} = h_{NC} \cdot A \cdot (T_w - T_{air}) + k_{air} \cdot A \cdot (T_w - T_{air}) + [\text{mixing of hot and cold air flow streams}]$$

The mixing of hot and cold air flow streams inside of the trailer was modeled by computing the air velocities in the x, y and z directions.

$$G_{r_L} = \frac{g \cdot \beta \cdot \Delta T \cdot L_c^3}{\nu^2} \approx Re^2 = \left(\frac{V \cdot L_c}{\nu} \right)^2 \Rightarrow V = \sqrt{g \cdot \beta \cdot \Delta T \cdot L_c}$$

$$\text{If } \rho_{air} = f(T) \Rightarrow V = \sqrt{\frac{\Delta \rho}{\rho} \cdot g \cdot L_c}$$

Lc - characteristic length of the geometry [m] (for the z direction, Lc is the wall height and for x and y Lc is A/P).

The mass that enters and exits a specific volume element in the container is computed using:

$$M_{air.i} = V_i \cdot A_{s.i} \cdot \rho_i$$

The final temperature value of each element due to air mixing is given by:

$$T_i = \frac{\sum M_i T_i}{\sum M_i}$$

$$M_C \cdot C_{p_C} \cdot \frac{\Delta T_C}{dt} = \underbrace{h_{NC} \cdot A \cdot (T_{air} - T_C)}_{\text{Natural convective heat transfer}} + \underbrace{\frac{k_C \cdot A \cdot (T_{air} - T_C)}{\Delta x}}_{\text{Conductive heat transfer}}$$

Natural convective
heat transfer

Conductive
heat transfer



Exterior dimensions:

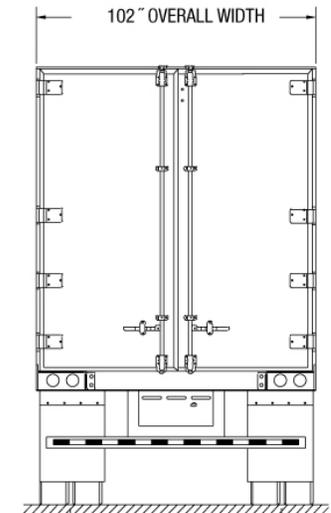
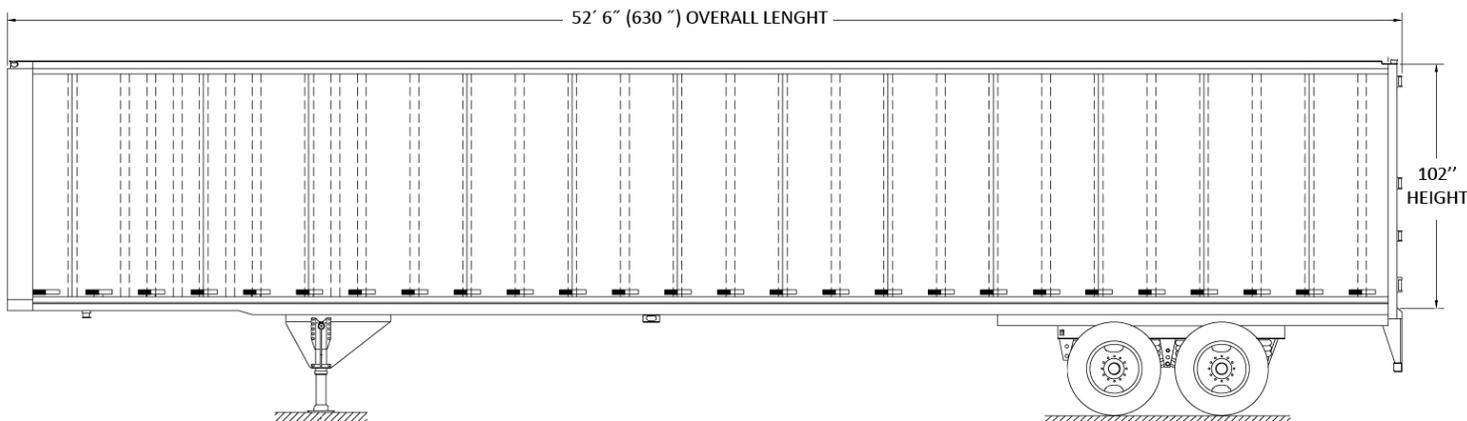
- Length 53 ft (630 in)
- Width 8.5 ft (102 in)
- Height 8.5 ft (102 in)

Side sheets:

- Aluminum
- Thermal conductivity 205 [W/m·K]
- Specific heat capacity 902 [J/kg·K]
- Density 2700 [kg/m³]
- 0.04 in thick

Insulation:

- Rigid polyurethane foam PUR/PIR
- Thermal conductivity 0.025 [W/m·K]
- Specific heat capacity 1500 [J/kg·K]
- Density 30 [kg/m³]
- Top, Left, Right and Back sides have a thickness of 1 in and the Front and Bottom side 2 in



Type: *Grocery Manufacturers' Association (GMA) pallet*

Dimensions:

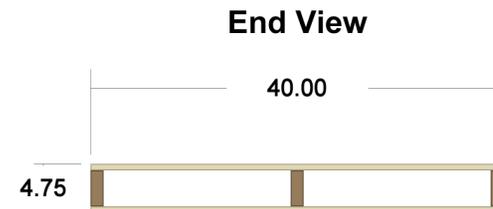
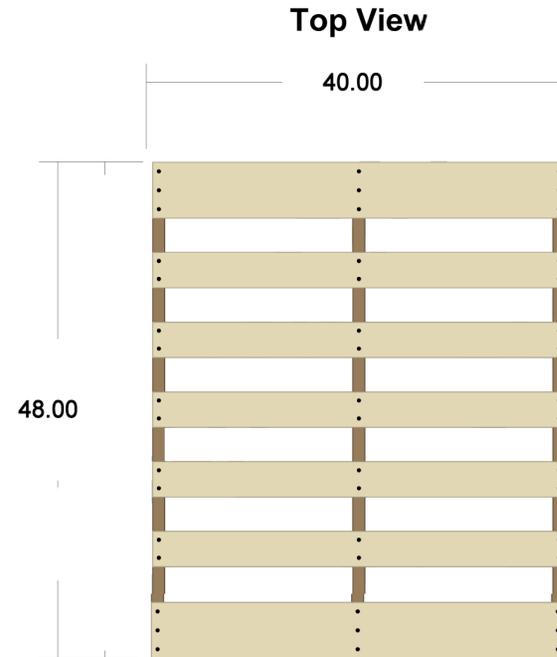
- Length: 48 in
- Width: 40 in
- Height: 4.75 in

Three types of loading were considered:

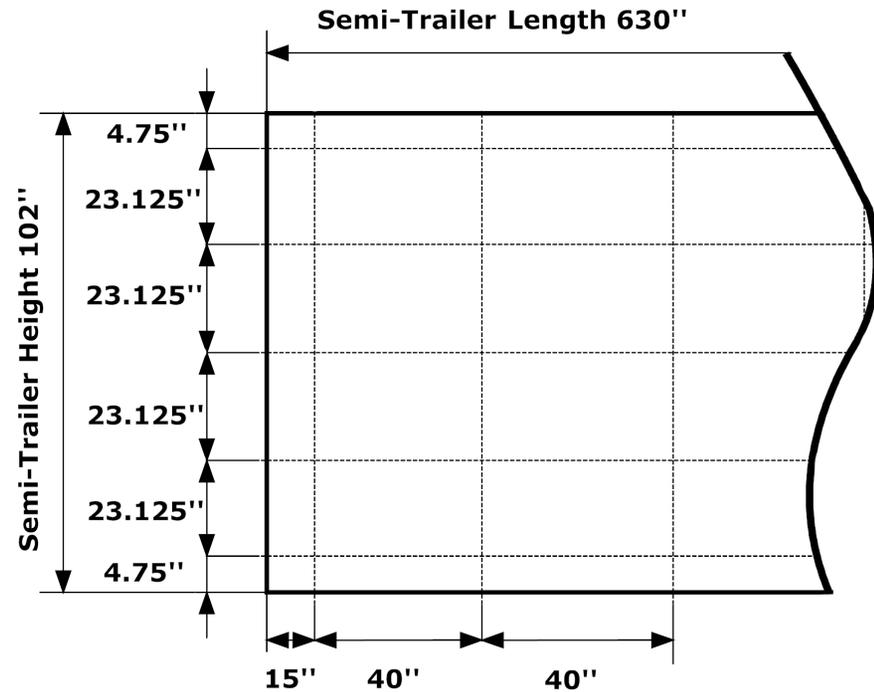
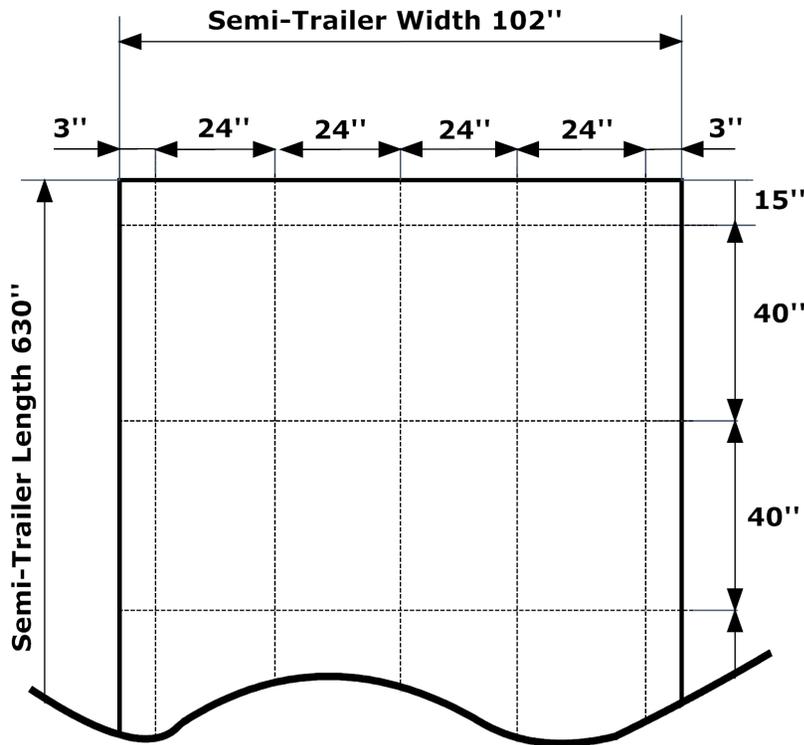
- Large (L) - Height of 46.25 in
- Extra-Large (XL) - Height of 69.375 in
- Jumbo (XXL) - Height of 92.25 in

The products on the pallets are considered to have the properties of water:

- Thermal conductivity 0.58 [W/m·K]
- Specific heat capacity 4180 [J/kg·K]
- Density 1000 [kg/m³]

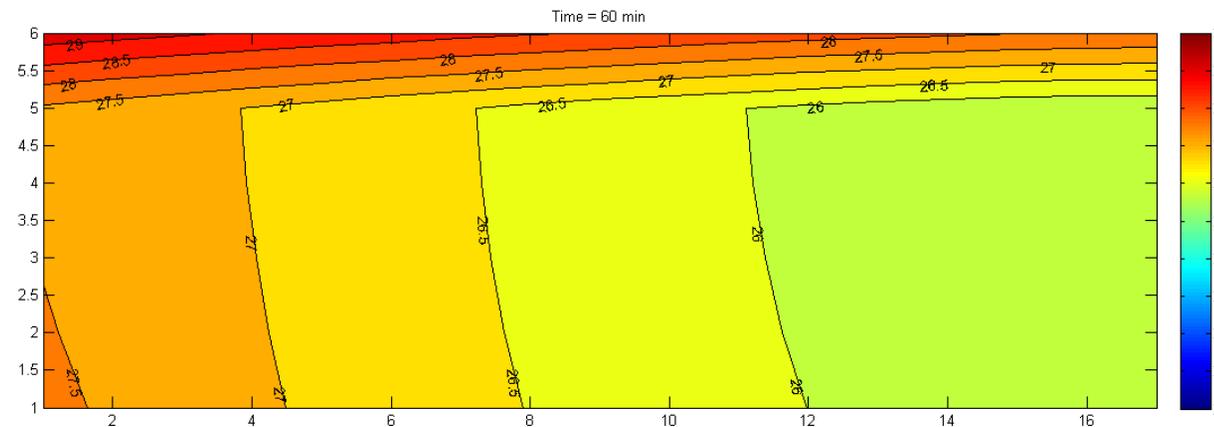
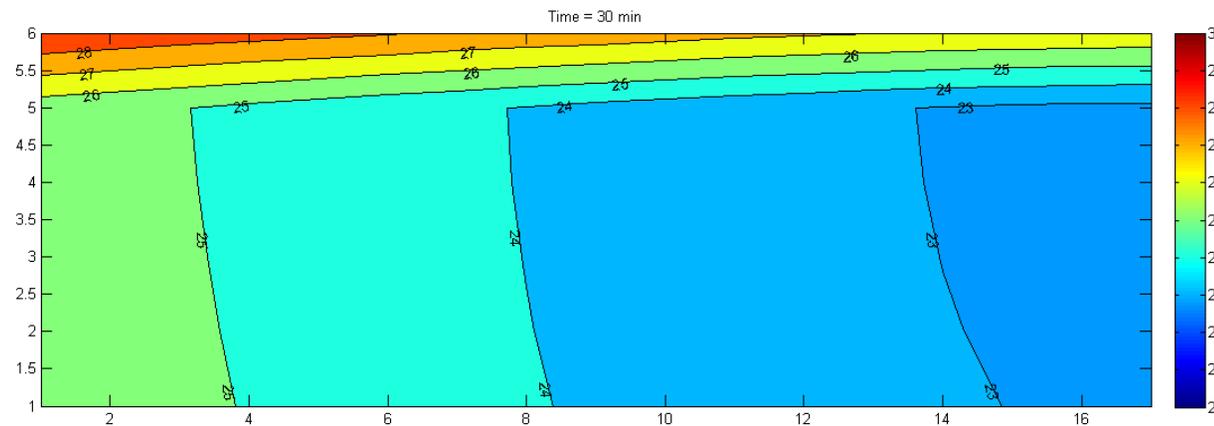
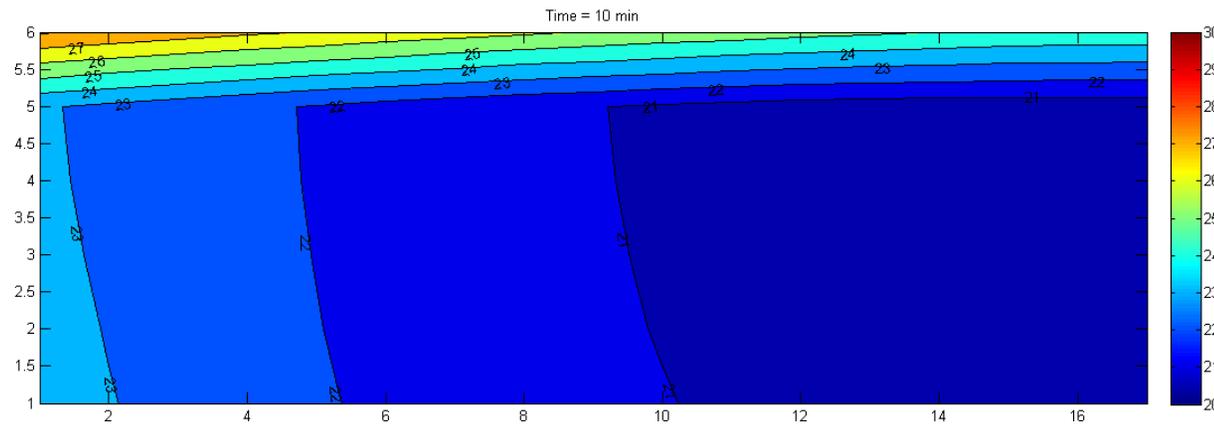


- The width of the trailer was divided into 6 segments.
- The length of the semi-trailer was divided into 17 segments.
- The height of the semi-trailer was divided into 6 segments.

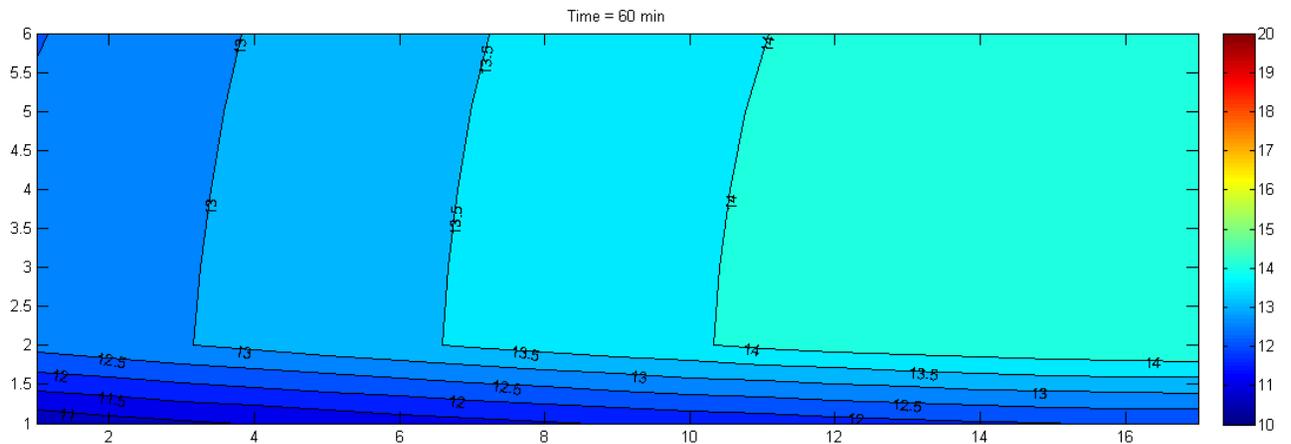
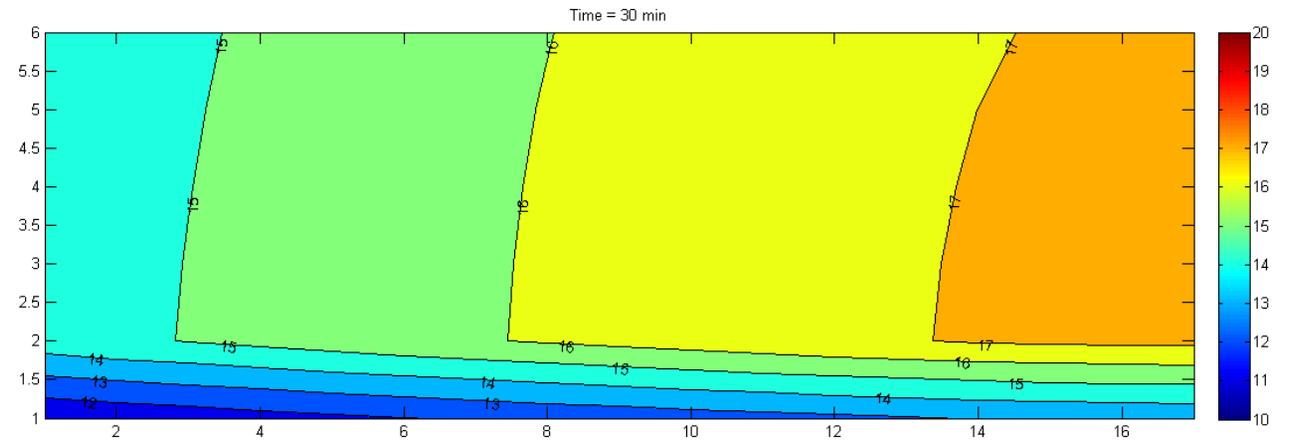
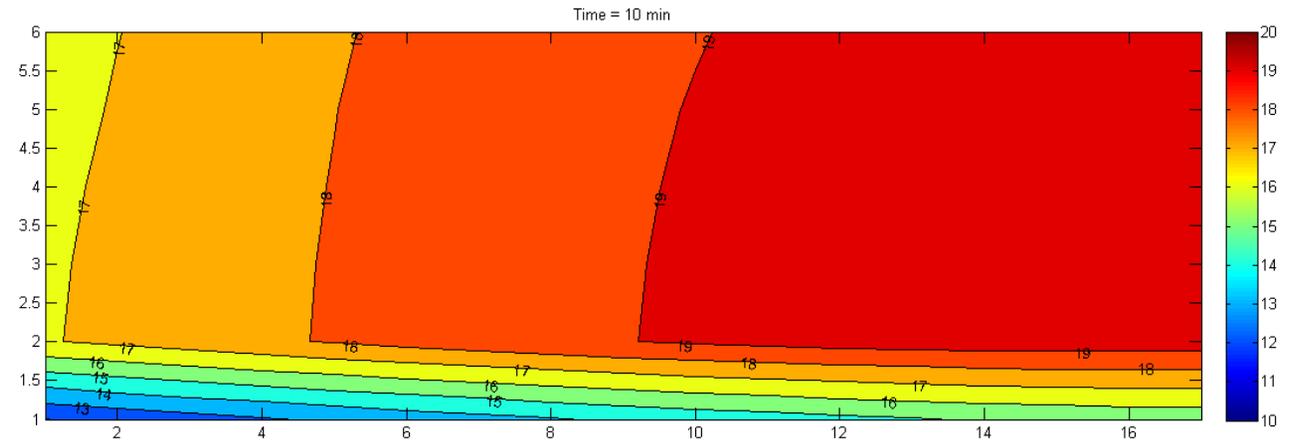




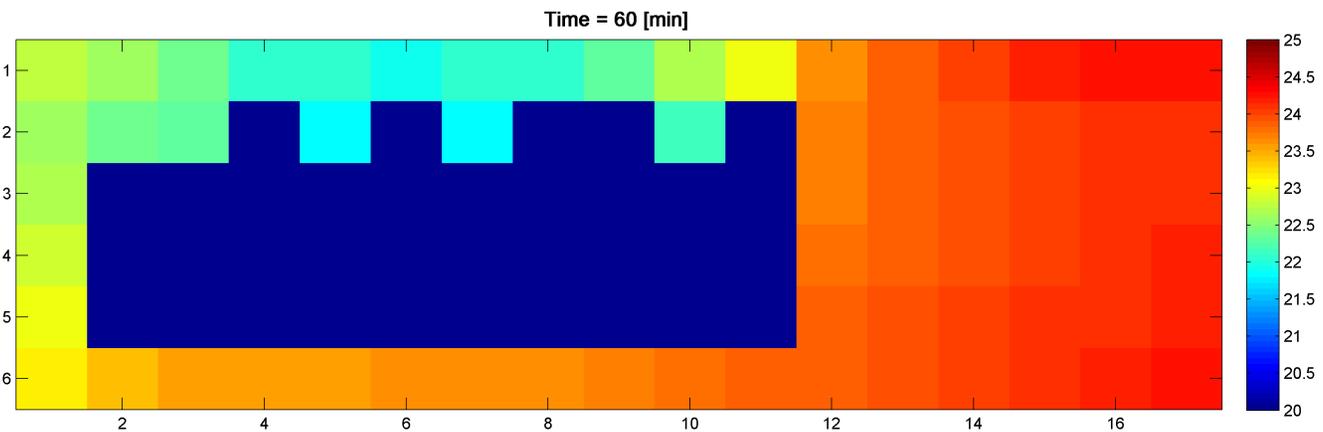
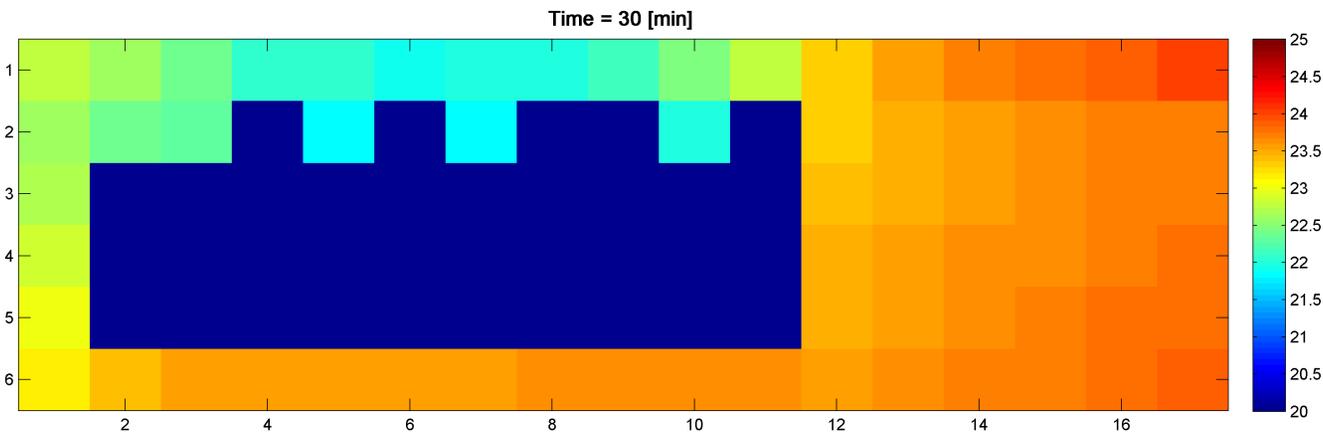
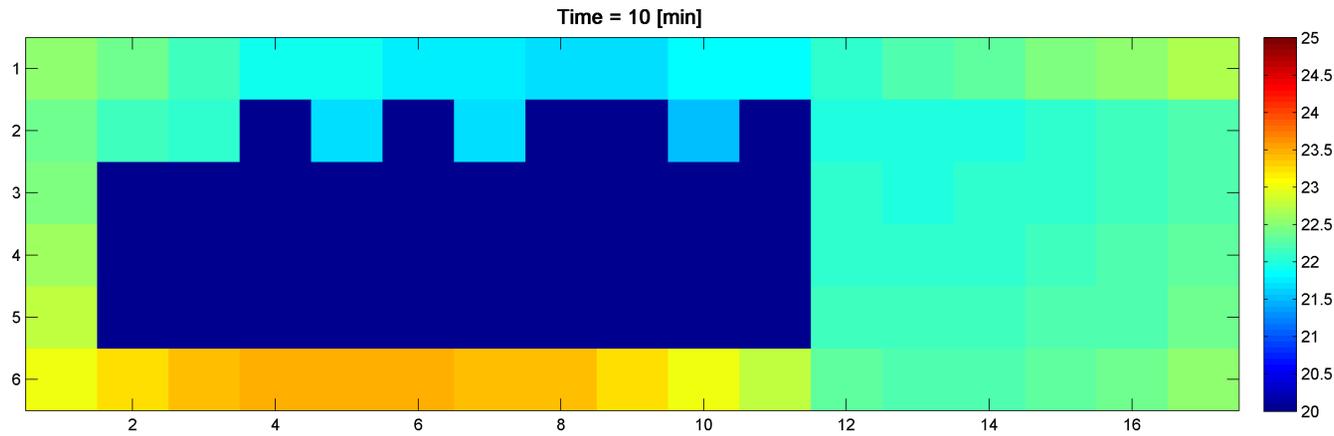
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Air temperature profile for the non-insulated semi-trailer at a speed of 45 miles/h, the outside temperature is 30°C and the inside initial temperature is 20°C.



Air temperature profile for the non-insulated semi-trailer at a speed of 45 miles/h, the outside temperature is 10°C and the inside initial temperature is 20°C.



Air temperature profile for the non-insulated semi-trailer at a speed of 60 miles/h, the outside temperature is 25°C and the inside initial temperature is 20°C, initial pallet temperature 20 °C.

Conclusions

The model can be used to simulate:

- shipping containers of all sizes
- insulated and non-insulated
- loaded with different types of pallets

Future work:

- Model validation with experimental data.
- Integrate the heat transfer model in supply chain models.